# The influence of TIME on my teaching and research 

Stephan V Joubert

Tshwane UOT
June 29, 2016

## Pronunciation of names

- Many of my students in South Africa have trouble with the pronunciation of my surname. They have no trouble with "Steve", or "Stephan", but "Joubert" is another story. Variations I have heard
- I too have trouble with African names such as
- We all have a good laugh at some of my attempts until I point out that one of our politicians, first name "Tokyo", probably has the strangest surname for a white Afrikaner-Scot like me to get around:
- After I point out the literal "English" pronunciation, we all move forward.


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- People who have influenced my teaching and research
- My career before TIME
- My career after TIME
- Throughout I will discuss some low cost real-life classroom exercises for under- and post graduates that have pointed me in the direction of some high-level technical publications.
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- Human resource development
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- Carsten Schmidt who taught me to do my teaching in the CAS laboratory whenever possible (I still do this today during my one-day-a week class).


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- After touring Europe for 6 months, I started working as a teacher at Clapham High School in 1972, teaching general science, even though I was busy with an honours degree in mathematics at that time.
- In 1975 I commenced my career at Tshwane University of Technology (TUT), teaching undergraduate physics, moving to the Department of Mathematics and Statistics (DMS) in 1981 and retiring there as "Professor Emeritus" in 2014 (40 years of service and still keeping my hand in).
- I received my DSc (Rings and R-modules) in 1984 from the University of Pretoria. Because TUT, as a university of technology (UOT), places a strong emphasis on the education of technologists, I was a "fish out of water".


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- My colleagues mostly had four-year bachelor's degrees (or an equivalent degree) and so a research environment in the DMS was almost non-existent.
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- In this way we gradually extend our knowledge of what we were teaching to research problems based on what our colleagues and students from other departments at TUT and the South African CSIR were doing
- At the turn of the century TUT introduced its own masters and doctoral programmes based on the USM model.
- Equipped with our newly acquired knowledge, my colleagues and I started publishing in "Mathematics Education" and other technical journals as well as attending conferences (education and tecthnicalall).


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- In 1997, as a first step towards doing some research in applied mathematics I advertised my willingness to help any colleagues who needed mathematical modeling for their own research.
- I was soon approached to give a series of lectures as part of an undergraduate course in explosive technology where I rubbed shoulders with a colleague (in the Department of Explosives Technology) doing her masters degree.
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## BAM friction meter

- These tiles were designed to be used in the Bundesanstalt fur Materialprufung or BAM friction meter.
- A small amount of explosive is spread onto the rough surface of the tile and a blunt pin is dragged across the rough surface. The amount of force needed to ignite the explosive can be varied as illustrated above.


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- To manufacture tiles for the South African explosives market with a prescribed mean roughness value $R_{a}=18.5 \mu \mathrm{~m}$, she used ceramic clay and extruded tiles with flat surfaces.
- Then, using a drill bit of radius say $r=180 \mu m$, she produced parallel grooves on one surface before baking the tiles.


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## Design of ceramic tiles (continued)

- The function $f(x)$ models the surface displacement above the $x$-axis.

- The arithmetic mean roughness value (ISO/DIS 4768/1) of a surface is defined as

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- Prescribing that $r=180 \mu \mathrm{~m}$ and $R_{a}=18.5 \mu \mathrm{~m}$ and solving a transcendental equation using DERIVE yielded

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k=263.9 \mu \mathrm{~m}
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\#13: [264, 263.883, 263.882, 263.882, 263.882, 263.882, 263.882]
Command: Guthor Build Calculus Declare Expand Factor Help Jump solve Manage options Plot Quit Remove Simplify Iransfer Unremove moUe Window approx
Enter option
Approx(\#12) C:\DERIUE3.05\CERAM Free:84\% Derive Algebra

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## Spin-offs

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- I obtained my first applied mathematics publication in the Harare conference proceedings published in the ISI accredited International Journal of Management and Systems.
- To date, the strategy of encouraging my colleagues and other students to publish and improve their qualifications has resulted in
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## My transition to TIME (continued)

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## Second ICTM, Crete 2002: The bouncing soccer ball (with Anton van Wyk)

- Equipment needed:
- Plastic soccer ball
- Kitchen or chemical scale
- Bathroom scale
- Ruler
- Measuring tape or metre rule
- Any CAS
- Typically our students have two semesters of calculus before they register for an introductory ODE course.
- Once the concept of second order ODEs with constant coefficients has been reached and a few easy exercises have been done, we introduce the idea of a spring-damper pair, with the accompanying concepts of over damped, under damped and critically damped free motion.


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- During this period of conceptual development, we give small groups of students a soccer ball. Initially they play with it before we start some measurements.


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## The bouncing soccer ball (continued)

- The ODE to consider solving for the simplified model of one bounce is

$$
M \ddot{x}+B \dot{x}+K x=0
$$



- A kitchen scale is used to measure the mass of the ball $M$.
- To measure the spring constant $K$, place the ball on top of a bathroom scale.
- Push down and hold the position at say a reading of 20 kg on the scale.
- Measure the ball compression with a ruler.


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## The bouncing soccer ball (continued)



- We found the compression distance to be approximately $\Delta x=6 \mathrm{~cm}$.
- From Hooke's law the ball "spring constant" k may be obtained.

$$
\begin{aligned}
K & =\frac{M g}{\Delta x} \\
K & =\frac{20 \times 9.8}{0.06} \\
& \approx 3.3 \times 10^{3} \mathrm{Nm}^{-1}
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## The bouncing soccer ball (continued)

- Determining the damping coefficient $B$ of the rubber ball is more difficult. One cannot pull on the ball as one would on a shock absorber.



## The bouncing soccer ball (continued)

- With a little mathematical "trickery" one can use the coefficient of restitution of the ball instead, by measuring the height of drop $h_{1}$ and the height of bounce $h_{2}$ using a tape measure.



## Coefficient of restitution

- The coefficient of restitution $R$ of the ball is the ratio of the incident velocity before bounce to the reflection velocity after bounce.
- The high school formula

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v^{2}=2 g h
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is used to show that
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\begin{aligned}
R & =\sqrt{\frac{h_{2}}{h_{1}}} \\
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and this can be used to determine the damping coefficient $B$.

- Applying these measurements to the equation of motion of the ball considered as a spring-damper pair, the trajectory of the ball may be modelled mathematically using DERIVE or any suitable CAS.


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## New Zealand DELTA03: An affordable, realistic student model of a motor-vehicle suspension system



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## Motor-vehicle suspension system

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\begin{aligned}
f(t)-k x-\phi \dot{x}+K(X-x) & =m \ddot{x} \\
-\Phi \dot{X}-K(X-x) & =M \ddot{X} \\
x(0)=X(0)=0 \text { and } \dot{x}(0) & =\dot{X}(0)=0
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## Spin-off publications (SAJS 2001, 2004) and SA patent with Keith Hardie and Temple Fay

- I had our team (Temple Fay, Keith Hardie, Anton van Wyk, Ansie Greeff et cetera) thinking about bouncing soccer balls and vehicle suspension systems.
- While driving over a corrugated dirt road in the Pilansberg Game Reserve near Pretoria (close to Buffelspoort where TIME2008 was held), I asked "I wonder how corrugations form". Keith went quiet for a while and then announced "I think I know how".
- Shortly after this the articles "Dirt road Corrugations" and "Towards the interactive wheel" appeared in the accredited South African Journal of Science and
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By gervision of hhrmy Mart and fiedt [netprises. Inc.

## Wheel (continued)

- Stan Wagon improved on BC's wheel by making a square wheel bicycle that rides smoothly over a catenery road:



## Wheel (continued)

- Seriously, our patent outlines the design of a wheel with a tyre that resists the formation of dirt road corrugations by reducing the tangential vibrations of the surface of the tyre on the road. Our contention is that it is the tangential vibrations of the tyre and not the radial vibrations of the tyre and vehicle springs that cause corrugations.


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- Indeed, she found among other results that corrugations have a wavelength of about 40 cm and that the frequency of vibrations (about 42 Hz ) of a vehicle travelling at $60 \mathrm{~km} \mathrm{~h}^{-1}$ agrees well with the frequency (about 44 Hz ) of tangential vibrations of a rubber tyre while radial spring frequency (about 2 Hz ) vibrations are not even of the same order of magnitude as tangential vibrations.


## Visit-Me TIME, Vienna, 2002: Dimension Analyses (with Temple Fay)

- Dimension analyses (DA) is folklore, but DA that results in a system of say three simultaneous equations with five unknowns is not readily understood by undergraduates at a UOT.
- The advent of CAS has made such analyses possible for our undergraduates.
- As an example of DA, consider that the period of small oscillation $T$ of a bob mass $M$ on a massless rod of length $\ell$ is known to be

$$
T \approx 2 \pi \sqrt{\frac{\ell}{g}}
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- In order to derive this formula, consider that the period of small oscillation $T$ may depend on the initial angular displacement $\theta_{0}$, the length of the (massless) rod $\ell$, mass of the bob $M$ and gravitational acceleration $g$.


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## Dimension Analyses

- In order to derive this result using DA, we form the dimensionless product

$$
\Pi_{G e n}=T^{x_{1}} \times \theta_{0}^{x_{2}} \times \ell^{x_{3}} \times M^{x_{4}} \times g^{x_{5}} .
$$

- Hence we obtain the system of linear equations:

$$
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\mathrm{m}: & 0 x_{1}+0 x_{2}+x_{3}+0 x_{4}+x_{5}=0 \\
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- Solving this system using DERIVE $®$ ) yields a general solution:

$$
\left(\begin{array}{l}
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\left(\begin{array}{l}
x_{1} \\
x_{2} \\
x_{3} \\
x_{4} \\
x_{5}
\end{array}\right)=a\left(\begin{array}{r}
1 \\
0 \\
-\frac{1}{2} \\
0 \\
\frac{1}{2}
\end{array}\right)+b\left(\begin{array}{l}
0 \\
1 \\
0 \\
0 \\
0
\end{array}\right), \quad a, b \in \mathbb{R}
$$

## Fundamental solutions

- Hence we see that $x_{4}=0$ and so mass does not play a role! It is obvious that we have two fundamental solutions:

$$
\left(\begin{array}{r}
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\end{array}\right) \text { and }\left(\begin{array}{c}
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\end{array}\right)
$$

- This means essentially that we have two non dimensional products $\Pi_{1}$ and $\Pi_{2}$ such that

$$
\Pi_{1}=T \times\left(\theta_{0}\right)^{0} \times \ell^{-1 / 2} \times m^{0} \times g^{1 / 2}=T \sqrt{\frac{g}{\ell}}
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## Buckingham's Theorem

- Buckingham's theorem yields

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T=\frac{\Psi\left(\theta_{0}\right)}{\left(\frac{g}{\ell}\right)^{1 / 2}}=\Psi\left(\theta_{0}\right) \times \sqrt{\frac{\ell}{g}}
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- where $\Psi$ is a function of $\theta_{0}$. An experiment can be conducted using small oscillations that shows that

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- Here we offer a simple DERIVE procedure that provides a measure of the accuracy of the Runge-Kutta order 4 DERIVE routine "RK4".
- Consider a linear system (although the routine works well with nonlinear systems):

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\begin{aligned}
& \ddot{x}-2 \dot{x}-3 x=-4 e^{t} \\
& x(0)=2, \quad \dot{x}(0)=0 .
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## Testing Accuracy of RK4

- Turn this IVP into a $2 \times 2$ system IVP as follows

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\begin{array}{ccc}
\dot{x} & = & y \\
\dot{y} & = & 2 y+3 x-4 e^{t} \\
x(0)=2, & & y(0)=0 .
\end{array}
$$

- Differentiating the second equation yields the equivalent $3 \times 3$ system

| $\dot{x}$ | $=$ | $y$ |
| :---: | :---: | :---: |
| $\dot{y}$ | $=$ | $z$ |
| $\dot{z}$ |  | $=$ |
| $x(0)=2$, | $y(0)=0$, | $z(0)=2$. |

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- The accuracy of the routine can be gauged by observing the difference (estimated error) between the numerical solutions.


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- Because we know the exact solution we can tabulate the actual error and compare this with the estimated error.
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| $t$ | Actual Error |
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| 1 | $9.061932091 \cdot 10^{-5}$ |
| 1.1 | 0.0001261640890 |
| 1.2 | 0.0001745509993 |
| 1.3 | ๑.0002403125765 |
| 1.4 | 0.0003295717135 |
| 1.5 | 0.0004505984321 |
| 1.6 | 0.0006145616139 |
| 1.7 | 0.0008365435693 |
| 1.8 | 0.001136909713 |
| 1.9 | 0.001543157466 |
| 2 | 0.002092412036 |


| $t$ | Estimated Error |
| :---: | :---: |
| 1 | $-5.943768169 \cdot 10^{-5}$ |
| 1.1 | $-8.297205588 \cdot 10^{-5}$ |
| 1.2 | -0.0001150529184 |
| 1.3 | -0.0001587005486 |
| 1.4 | -0.0002179952873 |
| 1.5 | -0.0002984487011 |
| 1.6 | -0.0004075046692 |
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## Spin offs

- Human resource development:
- Professor Dr. Ansie Greeff, my colleague and my doctoral student, has become well known in the field of mathematical biology. She has, with Temple Fay, developed a three competing species model with and without cropping.
- Ansie co-supervised with Temple Fay the doctoral studies of our colleague from the University of Pretoria Dr. Quay van der Hoff. Quay has made numerous presentation at TIME conferences. Temple Fay and I supervised Quay during her masters studies at TUT $\backslash$ USM.
- In an article published in the South African Journal of Science Ansie and I discussed the method dealt with in my TIME2004 presentation and adapted it to work on a three-species model, demonstrating that the model is accurate to at least three significant figures. The model will have no problem predicting populations measured in thousands over a 20-year period in the Kruger National Park.


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- Equipment needed:
- The qualitative effect of an earthquake or tornado moving down the edge of a bay or harbour can be observed in a classroom, with minimal apparatus.
- Indeed, anyone who causes a partially filled wineglass to "sing" (resonate) using a wet finger can observe the effect.
- If one carefully examines the surface of the liquid it is possible to see that four ridges or crests follow the finger around the rim. The wave patterns were already reported by Michael Faraday, in 1831 兰,

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## Singing wineglass

- Clearly the anti-node is below the energy source (the violin bow) when the glass is not rotating:

- The photograph on the left depicts a clockwise rotating glass and a static energy source (the finger).
- The Coriolis forces involved cause the anti-node to move to a fixed position away from the finger in the opposite direction to the rotation.
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## Bryan's effect

- This occurs because of what is known today as Bryan's Effect: "When a vibrating body rotates, the rotation pattern rotates away from its position of rest at a rate proportional to the rotation rate of the body".
- Indeed, Bryan's factor is the constant

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\eta=\frac{\text { Angular rate of the vibrating pattern }}{\text { Angular rate of the vibrating body }}
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## Navigation



> It turns out, after much investigation by our team, that this is Bryan's Effect modified by imperfections in the vibrating body such as mass and stiffness irregularities as well as anisotropic damping.

- For an ideal gyroscope without irregularities, if one knows the constant of proportionality (called Bryan's Factor) for a given substance, then one can use this effect for navigation.
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(1) Am. J. Phys (AJP) (2007). "A storm in a wineglass" with Temple Fay and Esme Voges
(2) IJMEST (2008). "The singing wineglass: an exercise in mathematical modelling" with Esme Voges
(3) AJP (2009). "Rotating structures and Bryan's effect" with Temple Fay and Michael Shatalov.
(9) Journal of Sound and Vibration (2009). "Free vibration of rotating hollow spheres containing acoustic media" with Michael Shatalov, Charlotta Coetzee and Igor Fedotov.


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## Malaga-TIME 2010, Spain: A CAS routine for obtaining eigenfunctions for Bryan's effect

- Consider a vibrating, slowly rotating annular disc:

- The polar coordinates $r$ and $\varphi$ of the position of rest $P$ of a vibrating particle in the annular disc of density $\rho$, height $h$, rotating slowly at rate $\varepsilon \Omega$. The vibrating particle at $P$ has radial displacement $u$ and tangential displacement $v$


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## Eigenfunctions

- If $U(r)$ and $V(r)$ the eigenfunctions associated with the eigenvalue of vibration $\omega^{2}$, assume that

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\begin{aligned}
u & =U(r)[C(t) \cos m \varphi+S(t) \sin m \varphi] \\
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where, $C(t)$ and $S(t)$ are to be determined and $m$ is the circumferential wave number.

- It has been demonstrated that $\omega^{2}=\frac{2 \int_{D}^{q} U V r d r}{\int_{P}^{q}\left(U^{2}+V^{2}\right) r d r}=\frac{1_{2}}{1_{0}}$.
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## Eigenfunctions(continued)

- Applying Newton's laws of motion to the mass element at point $P$, one arrives at two coupled PDE, and, assuming harmonic solutions, we arrive at a system of two coupled linear, ordinary differential equations (ODEs)

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\begin{gathered}
2 r^{2} U^{\prime \prime}+2 r U^{\prime}+\left(a \omega^{2} r^{2}-b\right) U+c r V^{\prime}-d V=0 \\
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- where, with With $\rho$ mass density, $E$ Young's modulus and $\mu$ Poisson's ratio we have

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\begin{gathered}
a=\frac{2\left(1-\mu^{2}\right) \rho}{E} ; b=m^{2}(1-\mu)+2 ; \\
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## Iteration process

－For a given substance and circumferential wave number $m$ ，the constants $a, b, c, d, e$ are known．If a value is assigned to $\omega$ ，then the NDSolve routine of Mathematica will produce a numerical solution for suitable boundary conditions．
－For a guess value of $\omega$ ，Mathematica $®$ ）stores $U$ and $V$ as interpolating functions．A new value for $\omega$ may now be generated by Mathematica $®$ ® via the known formula

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## TIME 2012 Tartu, Estonia: Using Fourier series to analyse mass imperfections in vibratory gyroscopes

- At this conference we considered once more the singing wineglass:

- Theoretically in an ideal wineglass we should be observing Bryan's Effect and the node should move away at a constant rotation rate, not be captured in a fixed position.
- By introducing density variations using a Fourier series, we are able to conduct a numerical experiment that predicts a", "capture effect".


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- Notice that, for instance

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- The kinetic energy of all of the particles in the disc is given by

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E_{k} \approx \frac{h}{2} \int_{0}^{2 \pi} \int_{p}^{q} \rho(\varphi)\left[\left(\dot{u}^{2}+\dot{v}^{2}\right)+2 \varepsilon \Omega(u \dot{v}-\dot{u} v)+2 \varepsilon \Omega \dot{v} r\right] r d r d \varphi .
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## Fourier series

- If we consider the density as a Fourier Series

$$
\rho(\varphi)=\rho_{0}\left(1+\varepsilon \sum_{k=1}^{\infty}\left(a_{k} \cos k \varphi+b_{k} \sin k \varphi\right)\right)
$$

- Then only the zeroth and $2 m^{\text {th }}$ harmonics play a roll in the kinetic energy expression and so we might as well choose

$$
\rho(\varphi):=\rho_{0}\left[1+\varepsilon \frac{4 I_{0}}{I_{3}}\left(\rho_{\mathrm{c}} \cos 2 m \varphi+\rho_{\mathrm{s}} \sin 2 m \varphi\right)\right] .
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- where for simplification purposes, we have set $a_{2 m}=4 \frac{10}{3} \rho_{c}$ and $b_{2 m}=4 \frac{1}{1_{3}} \rho_{\mathrm{s}}$ with $\rho_{\mathrm{c}}$ and $\rho_{\mathrm{c}}$ dimensionless constants that we will call the coefficients of the $2 m^{\text {th }}$ harmonics. The ratio $\frac{l_{0}}{l_{3}}$ is a ratio of definite integrals (we have already seen something similar with $\left.\frac{l_{2}}{I_{0}}=\omega^{2}=\frac{2 \int_{D}^{q} U V r d r}{\int_{P}^{q}\left(U^{2}+V^{2}\right) r d r}\right)$.


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## Kinetic energy

- With an eye on the Lagrange-Euler equations, we used Mathematica $®$ ® to calculate the various parts of the kinetic energy. For instance:

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\begin{aligned}
\operatorname{In}[128]= & \text { FullSimplify }[ \\
& \frac{\rho_{0} h}{2}\left(4 \in \frac{\mathbf{I}_{0}}{\mathbf{I}_{3}} \rho_{\mathrm{c}}\right) \\
& \int_{0}^{2 \pi} \int_{p}^{q} \operatorname{Cos}[2 m \varphi]\left(\left(\partial_{\mathrm{t}} u[r, \varphi, \mathrm{t}]\right)^{2}+\left(\partial_{\mathrm{t}} \mathrm{v}[r, \varphi, \mathrm{t}]\right)^{2}\right) \\
& r \mathrm{~d} r \mathrm{~d} \varphi, m \in \operatorname{Integers}]
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## Capture effect

- Considering the vibration pattern for the $m^{t h}$ circumferential wave number $m \Theta$, we have chosen ODE coefficients that show the following capture effect:

- The purple line indicates the ideal case with the precession angle $m \Theta$ for a slowly rotating disk changing linearly while the precession angle for a slowly rotating disk with light mass imperfections is "captured" periodically (the red/orange curve).
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## TIME 2014, Krems, Austria: Controlling mass-imperfections in vibratory gyroscopes

- If a disc gyroscope, with known Bryan's factor $\eta$, is mounted in a spacecraft and the vibration pattern of the gyroscope is observed, then a slow rate of rotation rate of the craft $\varepsilon \Omega$ may be measured via

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## Frequency split

- Mass imperfections cause beats (that can be heard in large VGs). Indeed, we have shown that the frequency of the beats is given by

$$
f=\frac{\varepsilon \omega \sqrt{\rho_{\mathrm{c}}^{2}+\rho_{\mathrm{s}}^{2}}}{2 \pi}
$$

where the coefficients of the $2 m^{t h}$ harmonics come from the Fourier series for the density

$$
\rho(\varphi):=\rho_{0}\left[1+\varepsilon \frac{4 I_{0}}{I_{3}}\left(\rho_{c} \cos 2 m \varphi+\rho_{\mathrm{s}} \sin 2 m \varphi\right)\right]
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## Equations of motion

- Indeed, the deviation from ideal behaviour can be observed from the equations of motion.
- Indeed, those of an ideal annular VG are

$$
\binom{\ddot{C}}{\ddot{S}}+\omega^{2}\binom{C}{S}=2 \eta \varepsilon \Omega\left(\begin{array}{cc}
0 & -1 \\
1 & 0
\end{array}\right)\binom{\dot{C}}{\dot{S}},
$$

- while those of a mass-imperfect annular VG are

$$
\binom{\ddot{C}}{\ddot{S}}+\omega^{2}\left(\begin{array}{cc}
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## Capacitor array

- Observe an annulus of thickness $h$ surrounded by an array of electronic plates each at a small distance $d$ from the cylindrical surface of the annulus. These plates, together with the cylindrical surface of the annulus approximate a "parallel plate capacitor" array:



## Small potential differences

- Assume that small potential differences $\sqrt{\varepsilon} V_{1}, \sqrt{\varepsilon} V_{2}, \sqrt{\varepsilon} V_{3}$ and $\sqrt{\varepsilon} V_{4}$ are maintained between the plate and the disc for capacitors numbered one to four respectively, where we use the small parameter $\varepsilon$ again to emphasise smallness.
- Assume that the other potential difference around the disc are $\frac{\pi}{2}$ periodic in the sense that capacitor number five has potential difference $\sqrt{\varepsilon} V_{1}$, capacitor number six has potential difference $\sqrt{\varepsilon} V_{2}$, et cetera.
- Consider the square of the electrical potential $\varepsilon V^{2}(\varphi)$ between the capacitors and the outer wall around the disc Let $d A$ be an infinitesimal part of the surface area on the outer cylindrical wall.
- Assume $\varepsilon V^{2}(\varphi)=0$ if there is no part of a plate covering the area $d A$ while $\varepsilon V^{2}(\varphi)=\varepsilon V_{1}^{2}$ if $d A$ is covered by the $1^{\text {st }}, 5^{\text {th }}, 9^{\text {th }}$ or $13^{\text {th }}$ plate, et cetera. An example of the situation is depicted in the following figure:


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## Fourier series



- Because of the periodicity involved with the potentials, we may determine a Fourier series for the function $V^{2}(\varphi)$ depicted in the figure as follows

$$
V^{2}(\varphi)=\frac{a_{0}}{2}+\sum_{n=1}^{\infty}\left(a_{n} \cos n \varphi+b_{n} \sin n \varphi\right) .
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$$

## Fourier coefficients

## We use Mathematica $®$ ® to determine the $a_{n}$ and $b_{n}$ :

$$
\begin{aligned}
& \ln [6]:=a_{n-}:= \\
& \frac{1}{\pi} \text { FullSimplify }\left[\int_{0}^{\Delta \varphi} \varepsilon \mathrm{V}_{1}^{2} \operatorname{Cos}[n \varphi] d l \varphi+\int_{\frac{\pi}{8}-\Delta \varphi}^{\frac{\pi}{8}+\Delta \varphi} \varepsilon \mathrm{V}_{2}^{2} \operatorname{Cos}[n \varphi] \mathrm{d} \varphi+\int_{\frac{\pi}{4}-\Delta \varphi}^{\frac{\pi}{4}+\Delta \varphi} \varepsilon \mathrm{V}_{3}^{2} \operatorname{Cos}[n \varphi] d l \varphi+\right. \\
& \int_{\frac{3 \pi}{8}-\Delta \varphi}^{\frac{3 \pi}{8}+\Delta \varphi} \varepsilon \mathrm{V}_{4}^{2} \operatorname{Cos}[n \varphi] \mathrm{dl} \varphi+\int_{\frac{\pi}{2}-\Delta \varphi}^{\frac{\pi}{2}+\Delta \varphi} \varepsilon \mathrm{V}_{1}^{2} \operatorname{Cos}[n \varphi] \mathrm{dl} \varphi+\int_{\frac{5 \pi}{8}-\Delta \varphi}^{\frac{5 \pi}{8}+\Delta \varphi} \varepsilon \mathrm{V}_{2}^{2} \operatorname{Cos}[n \varphi] d \varphi+\int_{\frac{3 \pi}{4}-\Delta \varphi}^{\frac{3 \pi}{4}+\Delta \varphi} \varepsilon \mathrm{V}_{3}^{2} \operatorname{Cos}[n \varphi] \mathrm{Cl} \varphi+ \\
& \int_{\frac{7 \pi}{8}-\Delta \varphi}^{\frac{7 \pi}{8}+\Delta \varphi} \varepsilon \mathrm{V}_{4}^{2} \operatorname{Cos}[n \varphi] d \varphi+\int_{\pi-\Delta \varphi}^{\pi+\Delta \varphi} \varepsilon \mathrm{V}_{1}^{2} \operatorname{Cos}[n \varphi] d \varphi+\int_{\frac{9 \pi}{8}-\Delta \varphi}^{\frac{9 \pi}{8}+\Delta \varphi} \varepsilon \mathrm{V}_{2}^{2} \operatorname{Cos}[n \varphi] d \varphi \varphi+\int_{\frac{5 \pi}{4}-\Delta \varphi}^{\frac{5 \pi}{4}+\Delta \varphi} \varepsilon \mathrm{V}_{3}^{2} \operatorname{Cos}[n \varphi] d \varphi+ \\
& \int_{\frac{11 \pi}{8}-\Delta \varphi}^{\frac{11 \pi}{8}+\Delta \varphi} \varepsilon \mathrm{V}_{4}^{2} \operatorname{Cos}[n \varphi] \mathrm{d} \varphi+\int_{\frac{3 \pi}{2}-\Delta \varphi}^{\frac{3 \pi}{2}+\Delta \varphi} \varepsilon \mathrm{V}_{1}^{2} \operatorname{Cos}[n \varphi] \mathrm{d} \varphi+\int_{\frac{13 \pi}{8}-\Delta \varphi}^{\frac{13 \pi}{8}+\Delta \varphi} \varepsilon \mathrm{V}_{2}^{2} \operatorname{Cos}[n \varphi] \mathrm{C} \varphi+ \\
& \left.\int_{\frac{7 \pi}{4}-\Delta \varphi}^{\frac{7 \pi}{4}+\Delta \varphi} \varepsilon \mathrm{V}_{3}^{2} \operatorname{Cos}[n \varphi] \mathrm{dl} \varphi+\int_{\frac{15 \pi}{8}-\Delta \varphi}^{\frac{15 \pi}{8}+\Delta \varphi} \varepsilon \mathrm{V}_{4}^{2} \operatorname{Cos}[n \varphi] \mathrm{dl} \varphi+\int_{2 \pi-\Delta \varphi}^{2 \pi} \varepsilon \mathrm{~V}_{1}^{2} \operatorname{Cos}[n \varphi] \mathrm{d} \varphi\right] \\
& \text { Table }\left[a_{n},\{n, 0,14\}\right] \\
& \text { Out[7] }=\left\{\frac{8 \Delta \varphi \varepsilon\left(V_{1}^{2}+V_{2}^{2}+V_{3}^{2}+V_{4}^{2}\right)}{\pi}, 0,0,0, \frac{2 \varepsilon \operatorname{Sin}[4 \Delta \varphi]\left(V_{1}^{2}-V_{3}^{2}\right)}{\pi}, 0,0,\right. \\
& \left.0, \frac{\varepsilon \operatorname{Sin}[8 \Delta \varphi]\left(V_{1}^{2}-V_{2}^{2}+V_{3}^{2}-V_{4}^{2}\right)}{\pi}, 0,0,0, \frac{2 \varepsilon \operatorname{Sin}[12 \Delta \varphi]\left(V_{1}^{2}-V_{3}^{2}\right)}{2 \pi}, 0,0\right\} \\
& \ln [8]:=\mathrm{Table}\left[\mathrm{~b}_{\mathrm{n}},\{\mathrm{n}, 1,14\}\right] \\
& \operatorname{Out}[8]=\left\{0,0,0, \frac{2 \varepsilon \operatorname{Sin}[4 \Delta \varphi]\left(V_{2}^{2}-V_{4}^{2}\right)}{\pi}, 0,0,0,0,0,0,0, \frac{2 \varepsilon \operatorname{Sin}[12 \Delta \varphi]\left(-V_{2}^{2}+V_{4}^{2}\right)}{3 \pi}, 0,0\right\}
\end{aligned}
$$

## Equations of motion

- Recall that $\omega^{2}=\frac{l_{2}}{l_{0}}=\frac{2 \int_{p}^{q} U V r d r}{\int_{P}^{q}\left(U^{2}+V^{2}\right) r d r}$ and examine the equations of motion that include mass imperfections and electrical potentials:

$$
\begin{aligned}
& \binom{\ddot{C}}{\ddot{S}}+ \\
& \left.\begin{array}{rl}
\frac{1}{I_{0}}\left(\begin{array}{cc}
l_{2}-\varepsilon k_{1}-\varepsilon\left[k_{2}+\rho_{c} I_{2}\right] \\
-\varepsilon\left[k_{3}+\rho_{s} I_{2}\right]
\end{array}\right. & -\varepsilon\left[k_{3}+\rho_{s} I_{2}\right] \\
I_{2}- & \varepsilon k_{1}+\varepsilon\left[k_{2}+\rho_{c} I_{2}\right]
\end{array}\right)\binom{C}{S}\binom{\dot{C}}{\dot{S}} \\
& =2 \eta \varepsilon \Omega\left(\begin{array}{cc}
0 & -1 \\
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\end{array}\right)
\end{aligned}
$$

- where

$$
\begin{aligned}
& k_{1}=\frac{4 \Delta \varphi h q \epsilon_{0} U^{2}(q)}{\pi d^{3}}\left(V_{1}^{2}+V_{2}^{2}+V_{3}^{2}+V_{4}^{2}\right) \\
& k_{2}=\frac{h q \epsilon_{0} \sin (4 \Delta \varphi) U^{2}(q)}{\pi d^{3}}\left(V_{1}^{2}-V_{3}^{2}\right)
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\left.\begin{array}{l}
I_{2}-\varepsilon k_{1}+\varepsilon\left[k_{3}+\rho_{s} I_{2}\right] \\
2
\end{array}\right)\left(\rho_{c} I_{2}\right]
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$$

## Negating mass imperfection

- Because $k_{2} \propto\left(V_{1}^{2}-V_{3}^{2}\right)$ and $k_{3} \propto\left(V_{2}^{2}-V_{4}^{2}\right)$, if we arrange capacitor voltage so that

$$
\varepsilon\left[k_{2}+\rho_{c} l_{2}\right]=0 \quad \& \quad \varepsilon\left[k_{2}+\rho_{c} l_{2}\right]=0
$$

- then the equations of motion reduce to

$$
\binom{\ddot{C}}{\ddot{S}}+\frac{I_{2}-\varepsilon k_{1}}{I_{0}}\binom{C}{S}=2 \eta \varepsilon \Omega\left(\begin{array}{cc}
0 & -1 \\
1 & 0
\end{array}\right)\binom{\dot{C}}{\dot{S}} .
$$

- Compare this with the equations of motion of an ideal annular gyroscope:

$$
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$$

where

$$
\omega^{2}=\frac{I_{2}}{r}=\frac{2 \int_{p}^{q} U V r d r}{q \quad 12}
$$

## Negative stiffness

- Hence, the capacitors have produced a gyroscope with mass imperfections that behaves "ideally" and is vibrating with a slightly reduced angular rate

$$
\omega^{*}=\sqrt{\frac{I_{2}-\varepsilon k_{1}}{I_{0}}}
$$

where $-\varepsilon k_{1}$ reduces the stiffness integral $I_{2}$ and is called negative stiffness. The term $-\varepsilon k_{1}$ will never disappear because $k_{1} \propto\left(V_{1}^{2}+V_{2}^{2}+V_{3}^{2}+V_{4}^{2}\right)$.

- An annular vibratory gyroscope manufactured by including this array of capacitors and manipulating them appropriately will be able to utilise Bryan's factor $\eta$ to determine the rotation rate $\varepsilon \Omega$ of the vehicle in which it is mounted using the formula

$$
\varepsilon \Omega=\frac{\text { Rate of rotation of the vibrating pattern of the gyroscope }}{\eta} .
$$

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\omega^{*}=\sqrt{\frac{l_{2}-\varepsilon k_{1}}{I_{0}}}
$$

where $-\varepsilon k_{1}$ reduces the stiffness integral $I_{2}$ and is called negative stiffness. The term $-\varepsilon k_{1}$ will never disappear because $k_{1} \propto\left(V_{1}^{2}+V_{2}^{2}+V_{3}^{2}+V_{4}^{2}\right)$.

- An annular vibratory gyroscope manufactured by including this array of capacitors and manipulating them appropriately will be able to utilise Bryan's factor $\eta$ to determine the rotation rate $\varepsilon \Omega$ of the vehicle in which it is mounted using the formula

$$
\varepsilon \Omega=\frac{\text { Rate of rotation of the vibrating pattern of the gyroscope }}{\eta} .
$$

